

Name (please print):

Signature:

Student number:

SOLUTIONS

**FACULTY OF SCIENCE  
YORK UNIVERSITY  
DEPARTMENT OF PHYSICS AND ASTRONOMY**

**PHYS 1800 03 – Engineering Mechanics  
Fall 2014**

**Final examination**

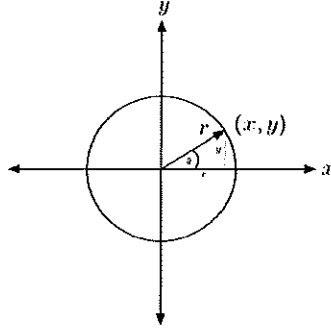
**Instructions:**

1. Print your name and student number on this page in the upper left corner.
2. Check that your exam contains 7 questions.
3. The total number of marks is 100.
4. Simple scientific calculators are allowed as aid.
5. Answer all questions in the space provided on this exam paper.
6. Formulae are provided on the last page.

1.	
2.	
3.	
4.	
5.	
6.	
7.	
Total	

1. (10 marks)

An object (mass  $m = 0.20$  kg) is moving with constant speed in a circle in the  $xy$  plane. The position of the object is given by the following vector:  $\mathbf{r} = 0.8 \cos(2.5 t)\mathbf{i} + 0.8 \sin(2.5 t)\mathbf{j}$ , where  $r$  is in meters and  $t$  is in seconds.



a. Determine the velocity (magnitude and direction) of the object at  $t = 2.0$  s.

$$\mathbf{v} = \frac{d\mathbf{r}}{dt} = -2.0 \sin(2.5 t)\mathbf{i} + 2.0 \cos(2.5 t)\mathbf{j},$$

$$\mathbf{v}(t = 2.0 \text{ s}) = -2.0 \sin[(2.5)(2.0)]\mathbf{i} + 2.0 \cos[(2.5)(2.0)]\mathbf{j} = 1.92\mathbf{i} + 0.567\mathbf{j} \text{ (m/s)}$$

$$v = \sqrt{(1.92)^2 + (0.567)^2} = 2.0 \text{ m/s}$$

$$\tan \theta = \frac{0.567}{1.92} \quad \theta = 16.45^\circ \approx 16^\circ$$

b. Determine the acceleration (magnitude and direction) of the object at  $t = 2.0$  s.

$$\mathbf{a} = \frac{d\mathbf{v}}{dt} = -5.0 \cos(2.5 t)\mathbf{i} - 5.0 \sin(2.5 t)\mathbf{j},$$

$$\mathbf{a}(t = 2.0 \text{ s}) = -5.0 \cos[(2.5)(2.0)]\mathbf{i} - 5.0 \sin[(2.5)(2.0)]\mathbf{j} = -1.42\mathbf{i} + 4.79\mathbf{j} \text{ (m/s}^2\text{)}$$

$$a = \sqrt{(-1.42)^2 + (4.79)^2} = 5.0 \text{ m/s}^2$$

$$\tan \theta = \frac{4.79}{-1.42} \quad \theta = -73.48^\circ \approx -74^\circ$$

c. What is the angle between velocity and acceleration vectors at  $t = 2.0$  s?

90 degrees

d. How long does it take the object to make 5.0 revolutions?

$$\omega = 2.5 \text{ Hz} \quad \frac{2\pi}{T} = 2.5 \quad T = 2.51 \text{ s}$$

$$\text{Time of 5 rev} = 5(2.51) = 12.6 \text{ s} \approx 13 \text{ s}$$

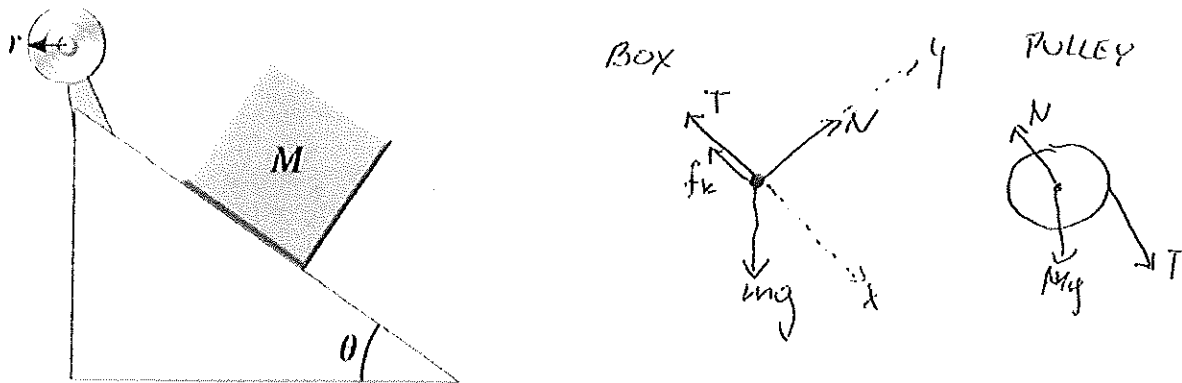
e. What is the magnitude of the force keeping the object in circular motion?

$$F = ma = (0.20 \text{ kg})(5.0 \text{ m/s}^2) = 1.0 \text{ N}$$

2. (12 marks)

A box of mass  $M = 12.0 \text{ kg}$  is attached to a cord that is wrapped around a pulley whose moment of inertia is  $I = 0.15 \text{ kg m}^2$  and radius  $r = 0.300 \text{ m}$ . The coefficient of kinetic friction between the box and the incline surface is  $\mu_k = 0.25$  and the angle  $\theta = 30.0^\circ$ .

- Draw a free-body force diagram for the box and the pulley.
- Determine the kinetic force of friction acting on the box.
- Determine the acceleration of the box, angular acceleration of the pulley and the tension in the cord.



$$b) f_k = \mu_k N = \mu_k mg \cos \theta = (0.25)(12.0)(9.81) \cos 30.0^\circ = 25.49 \text{ N} \approx 25 \text{ N}$$

c) BOX

$$\sum F_x = ma_x$$

$$mg \sin 30.0^\circ - T - 25.49 = 12.0 a$$

$$(12.0)(9.81) \sin 30.0^\circ - T - 25.49 = 12.0 a$$

$$33.37 - T = 12.0 a$$

PULLEY

$$\sum \tau = I \alpha$$

$$-Tr = I \left( -\frac{a}{r} \right)$$

$$0.300 T = (0.15) \left( \frac{a}{0.300} \right)$$

$$T = 1.67 a$$

$$33.37 - 1.67 a = 12.0 a$$

$$13.67 a = 33.37$$

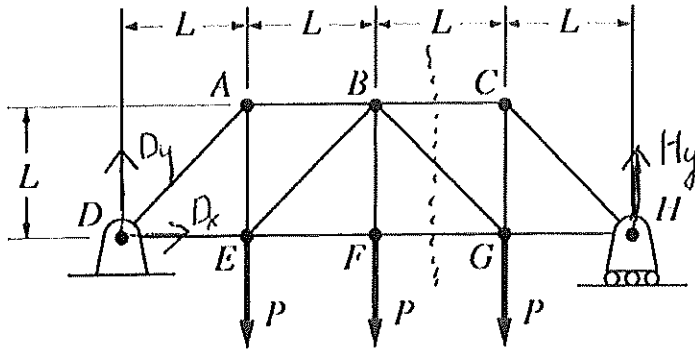
$$a = 2.44 \text{ m/s}^2 \approx 2.4 \text{ m/s}^2$$

$$T = (1.67)(2.44) = 4.1 \text{ N}$$

$$\alpha = \frac{a}{r} = \frac{2.44}{0.300} = 8.1 \text{ rad/s}^2$$

3. (12 marks)

Determine the forces in the following truss members: BC, BG and FG. State whether each member is in tension or compression. Assume that  $L = 3.00 \text{ m}$  and  $P = 5000.0 \text{ N}$ . The roller is frictionless.



$$\sum F_x = 0$$

$$D_x = 0$$

$$\sum F_y = 0$$

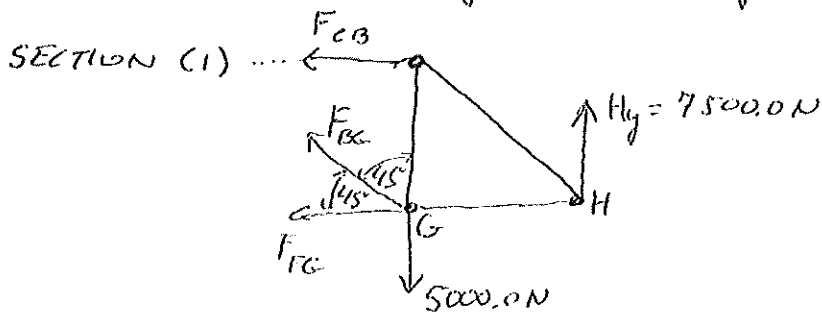
$$D_y + H_y - 15000.0 = 0$$

$$\sum M_D = 0$$

$$\sum M_D = -(5000.0)(3.00) - (5000.0)(6.00) - (5000.0)(9.00) + H_y(12.0) = 0$$

$$H_y = 7500.0 \text{ N}$$

$$D_y = 15000.0 - H_y = 15000.0 - 7500.0 = 7500.0 \text{ N}$$



$$\sum M_G = (7500.0)(3.00) + F_{BC}(3.00) = 0 \Rightarrow F_{BC} = -7500.0 \text{ N (TENSION)}$$

$$\sum F_x = -F_{BG} \cos 45^\circ - F_{FG} - F_{BC} = 0$$

$$F_{FG} = +7500.0 - F_{BG} \cos 45^\circ$$

$$\sum F_y = F_{BG} \sin 45^\circ + H_y - 5000.0 = 0$$

$$F_{BG} \sin 45^\circ = 5000.0 - 7500.0$$

$$F_{BG} = -3535 \text{ N (COMPRESSION)}$$

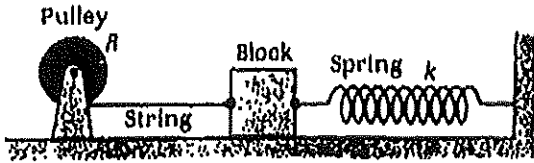
$$F_{FG} = +7500.0 - (-3535) \cos 45^\circ$$

$$F_{FG} = +5000.0 \text{ N (COMPRESSION)}$$

$$10000 \text{ N (TENSION)}$$

4. (10 marks)

A pulley of radius  $R = 0.070$  m and the moment of inertia  $I = 0.01$  kg m<sup>2</sup> is rotated clockwise until the spring is stretched  $d = 0.50$  m, and then released from rest. Determine the speed of the block (mass  $m = 4.3$  kg) when it passes through the position at which the spring is un-stretched. The spring constant is  $k = 235$  N/m. The coefficient of kinetic friction between the block and the surface is  $\mu_k = 0.15$ . The pulley is frictionless.



$$\Delta K + \Delta U = W_{nc}$$

$$\Delta K = K_f - K_i = \left( \frac{1}{2} m v^2 - 0 \right) + \left( \frac{1}{2} I \omega^2 - 0 \right) = \frac{1}{2} m v^2 + \frac{1}{2} I \frac{v^2}{R^2}$$

$$\Delta U_s = U_{sf} - U_{si} = 0 - \frac{1}{2} k d^2 = -\frac{1}{2} k d^2$$

$$W_{nc} = f_k d \cos 180^\circ = -\mu_k m g d$$

$$\frac{1}{2} m v^2 + \frac{1}{2} I \frac{v^2}{R^2} - \frac{1}{2} k d^2 = -\mu_k m g d$$

$$v^2 \left( \frac{m}{2} + \frac{I}{2R^2} \right) = \frac{1}{2} k d^2 - \mu_k m g d$$

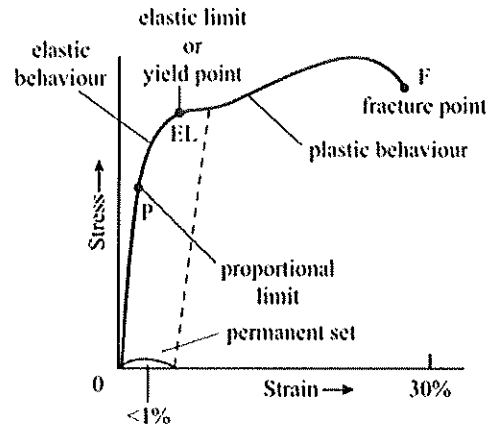
$$v = \sqrt{\frac{\frac{1}{2} k d^2 - \mu_k m g d}{\frac{m}{2} + \frac{I}{2R^2}}}$$

$$v = \sqrt{\frac{\frac{1}{2} (235) (0.50)^2 - (0.15) (4.3) (9.81) (0.50)}{\frac{4.3}{2} + \frac{0.01}{2(0.070)^2}}$$

$$v = 2.9 \text{ m/s.}$$

5. (8 marks)

Draw a typical stress-strain dependence and explain the meaning of the following terms:



**Proportional limit**

The maximum stress for which stress-strain graph is linear (point P in the graph)

**Yield point**

The maximum stress for which the material remains elastic (when stress is removed the material returns to original length and shape).

**Permanent set**

The amount of deformation (strain) that remains when the stress is removed.

**Plastic deformation**

Permanent deformation that occurs when for stress applied is larger than yield point

**Ultimate strength**

The maximum stress on the stress-strain graph

**Engineering stress**

The force per unit area of the original cross-section of the material

**True stress**

The force per actual cross-section as the material is deformed. True stress continues to increase until the material fractures because the actual cross-section area decreases

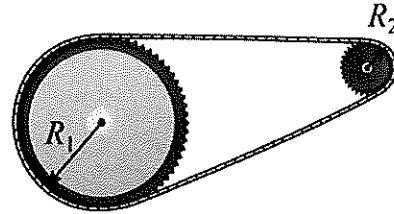
6. (18 marks)

Answer the following six questions.

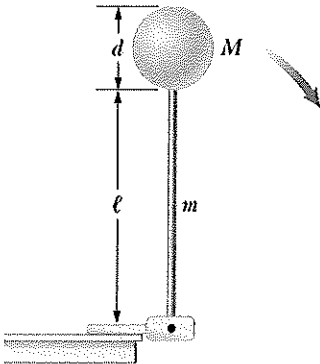
a. Two gears ( $R_1 = 52 \text{ cm}$  and  $R_2 = 17 \text{ cm}$ ) mounted on fixed axes are connected by a belt. The smaller gear is driven by a motor that causes it to spin at  $7.0 \text{ rad/s}$ . The belt does not slip. Determine the angular speed of the larger gear.

$$\omega_1 R_1 = \omega_2 R_2$$

$$\omega_1 = \frac{\omega_2 R_2}{R_1} = \frac{(7.0)(0.17)}{(0.52)} = 2.3 \text{ rad/s}$$

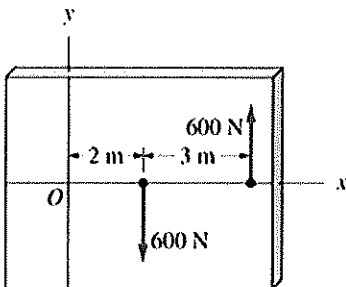


b. Determine the moment of inertia of the object (shown below) about an axis perpendicular to the page, running through the black dot. The object consists of a thin rod (mass  $m$ , length  $\ell$ ) and a solid sphere (mass  $M$ , diameter  $d$ ). Express the moment of inertia in terms of  $m$ ,  $\ell$ ,  $M$  and  $d$ .



$$I = I_{\text{rod}} + I_{\text{sphere}} = (1/3)mL^2 + (2/5)M(d/2)^2 + M(d + L)^2 \text{ (Attention: The parallel-axis theorem was omitted in 2015)}$$

c. Determine the magnitude and direction of the moment of the couple for forces for the situation shown below.

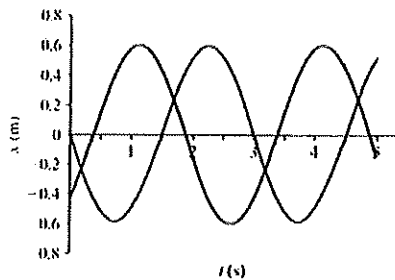


$$M = -(600)(2) + (600)(5) = 1800 \text{ Nm}$$

DIRECTION: PARALLEL TO Z-AXIS

Continued on the next page...

d. What is the phase difference between the two oscillators?



$$x = A \cos(\omega t + \phi)$$

(1) SUBSTITUTING

$$t=0, x=0$$

$$0 = 0.6 \cos \phi_1$$

$$\phi_1 = +\frac{\pi}{2} \text{ RAD } (+90^\circ)$$

$$\text{OR } -\frac{3\pi}{2} \text{ RAD } (-270^\circ)$$

(2) SUBSTITUTING

$$t=0, x=-0.4$$

$$-0.4 = 0.6 \cos \phi_2$$

$$\phi_2 = -2.3 \text{ RAD } (-132^\circ)$$

$$\text{OR } +4.0 \text{ RAD } (+228^\circ)$$

$$\Delta\phi = \phi_1 - \phi_2 = \frac{\pi}{2} - (-2.3) = 3.9 \text{ RAD } (222^\circ)$$

$$\text{OR } (2\pi - 3.9) = 2.4 \text{ RAD } (138^\circ)$$

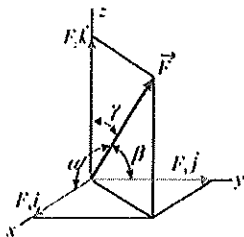
e. A travelling wave is described by the following equation:  $D(x,t) = 0.2 \sin(0.30x - 5.0t + 0.4)$ .

What is the speed of the wave?

$$v = \frac{\omega}{k}$$

$$v = \frac{5.0}{0.2} = 25 \text{ m/s}$$

f. Determine the angles  $\alpha$ ,  $\beta$  and  $\gamma$  between the force  $F = 1200i + 800j - 1500k$  and the  $x$ -,  $y$ -, and  $z$ -axes.



$$F = \sqrt{(1200)^2 + (800)^2 + (-1500)^2} = 2081$$

$$\cos \alpha = \frac{F_x}{F} = \frac{1200}{2081} \Rightarrow \alpha = 55^\circ$$

$$\cos \beta = \frac{F_y}{F} = \frac{800}{2081} \Rightarrow \beta = 67^\circ$$

$$\cos \gamma = \frac{F_z}{F} = \frac{-1500}{2081} \Rightarrow \gamma = 136^\circ$$

7. (30 marks)

Answer the following five questions.

a. An engine pulls a 1000.0 kg crate from rest to 20.0 m/s in 5.00 s with constant acceleration. The force of kinetic friction between the crate and the ground is  $f_k = 500.0$  N. How much work is done by the engine?

$$W = \Delta K$$

$$W_{\text{ENGINE}} + W_{\text{FRICTION}} = \frac{1}{2} m v^2$$

$$W_{\text{ENGINE}} = \frac{1}{2} m v^2 - f_k d \cos 180^\circ = \frac{1}{2} m v^2 + f_k d$$

TO FIND  $d$ :  $v = v_0 + at$ ,  $v_0 = 0$

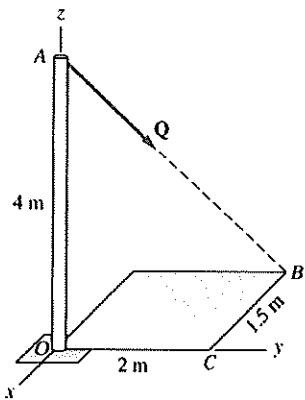
$$a = \frac{v}{t} = \frac{20.0}{5.00} = 0.40 \text{ m/s}^2$$

$$d = \frac{1}{2} at^2 = \frac{1}{2} (0.40) (5.00)^2 = 5.00 \text{ m}$$

$$W_{\text{ENGINE}} = \frac{1}{2} (1000.0) (20.0)^2 + (500.0) (5.00) = 31000 \text{ J}$$

$$P = \frac{W_{\text{ENGINE}}}{t} = \frac{31000 \text{ J}}{5.00 \text{ s}} = 6200 \text{ W}$$

b. Determine the moment of the force  $Q$  about point  $O$ . The magnitude of  $Q$  is 100.0 N.



$$\vec{Q} = Q \hat{u}_{AB}$$

$$\hat{u}_{AB} = \frac{\vec{AB}}{AB} = \frac{-1.5\hat{i} + 2\hat{j} - 4\hat{k}}{\sqrt{(-1.5)^2 + 2^2 + (-4)^2}} =$$

$$\hat{u}_{AB} = -0.318\hat{i} + 0.424\hat{j} - 0.847\hat{k}$$

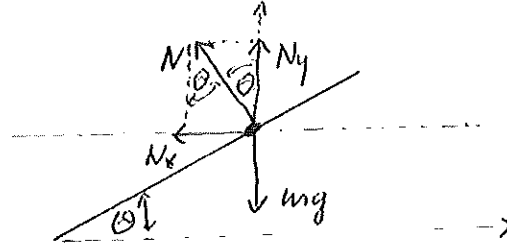
$$\vec{Q} = 100.0 (-0.318\hat{i} + 0.424\hat{j} - 0.847\hat{k})$$

$$\vec{Q} = -31.8\hat{i} + 42.4\hat{j} - 84.7\hat{k}$$

$$M_O = \vec{r}_{OA} \times \vec{Q} = \begin{vmatrix} \hat{i} & \hat{j} & \hat{k} \\ 0 & 0 & 4 \\ -31.8 & 42.4 & -84.7 \end{vmatrix} = -1696\hat{i} - 127.2\hat{j} + 0\hat{k}$$

Continued on the next page...

c. An engineer is given a task to calculate the slope of a curved banked portion of a highway to prevent cars from slipping off the road on icy day when friction is almost zero. Assume that the radius of the curved banked road is  $R = 190.0 \text{ m}$  and that the speed limit is  $v = 100.0 \text{ km/h}$ . Draw a free-body force diagram for the car on a banked road and determine the bank angle  $\theta$  that prevents cars sliding off the road.



$$\begin{aligned} \sum F_x &= m a \\ N_x &= m \frac{v^2}{R} \\ N \sin \theta &= m \frac{v^2}{R} \end{aligned}$$

$$\begin{aligned} \sum F_y &= 0 \\ N \cos \theta - m g &= 0 \\ N &= \frac{m g}{\cos \theta} \end{aligned}$$

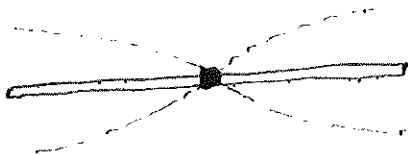
$$\frac{m g \sin \theta}{\cos \theta} = m \frac{v^2}{R}$$

$$\tan \theta = \frac{v^2}{R g}$$

$$v = (100.0 \text{ km/h}) \left( \frac{1 \text{ h}}{3600 \text{ s}} \right) \left( \frac{1000 \text{ m}}{1 \text{ km}} \right) = 16.7 \text{ m/s}$$

$$\tan \theta = \frac{(16.7)^2}{(190.0)(9.81)} \Rightarrow \theta = 8.5^\circ$$

d. An aluminum rod  $L = 1.60 \text{ m}$  long is held at its center. It is hit with a hammer to set up a longitudinal vibration (like you did in the lab). The density of aluminum is  $\rho = 2700.0 \text{ kg/m}^3$  and Young's modulus  $Y = 7.0 \times 10^{10} \text{ Pa}$ . Calculate the speed of longitudinal waves in the rod and the wavelength and frequency of the fundamental standing wave (first harmonic).



$$v = \sqrt{\frac{Y}{\rho}} = \sqrt{\frac{7.0 \times 10^{10}}{2700}} = 5091 \text{ m/s}$$

$$\frac{\lambda}{2} = L \Rightarrow \lambda = 2L = 2(1.60) = 3.20 \text{ m}$$

$$v = \lambda f \Rightarrow f = \frac{v}{\lambda} = \frac{5091}{3.20} = 1590 \text{ Hz} \approx 1600 \text{ Hz}$$

THE END

$$x = x_0 + v_0 t + \frac{1}{2} a t^2 \quad v = v_0 + a t \quad v^2 = v_0^2 + 2a(x - x_0) \quad v_{avg} = \frac{1}{2}(v_0 + v) \quad (x - x_0) = \frac{1}{2}(v_0 + v)t$$

$$r(t) = x(t)\mathbf{i} + y(t)\mathbf{j} + z(t)\mathbf{k} \quad \mathbf{v}_{avg} = \frac{\Delta \mathbf{r}}{\Delta t} \quad \mathbf{v} = \frac{d\mathbf{r}}{dt} = \frac{dx}{dt}\mathbf{i} + \frac{dy}{dt}\mathbf{j} + \frac{dz}{dt}\mathbf{k} \quad \mathbf{a}_{avg} = \frac{\Delta \mathbf{v}}{\Delta t} \quad \mathbf{a} = \frac{d\mathbf{v}}{dt} = \frac{dv_x}{dt}\mathbf{i} + \frac{dv_y}{dt}\mathbf{j} + \frac{dv_z}{dt}\mathbf{k}$$

$$F = ma \quad f = \mu N \quad a_c = \frac{v^2}{r} \quad F_g = mg \quad F_g = \frac{GMm}{r^2} \quad F = -kx$$

$$\theta = \frac{s}{r} \quad \omega = \frac{d\theta}{dt} \quad \alpha = \frac{d\omega}{dt} \quad \Delta\theta = \omega_0 t + \frac{1}{2} \alpha t^2 \quad \omega = \omega_0 + \alpha t \quad \Delta\theta = \left(\frac{\omega_i + \omega_f}{2}\right)t$$

$$v = r\omega \quad \mathbf{v} = \boldsymbol{\omega} \times \mathbf{r} \quad a_c = \frac{v^2}{r} = r\omega^2 \quad \mathbf{a}_c = \boldsymbol{\omega} \times (\boldsymbol{\omega} \times \mathbf{r}) \quad a_t = \frac{dv}{dt} = r\alpha \quad \mathbf{a}_t = \boldsymbol{\alpha} \times \mathbf{r}$$

$$\boldsymbol{\tau} = \mathbf{r} \times \mathbf{F} \quad \boldsymbol{\tau} = I \boldsymbol{\alpha} \quad I_{CM}^{rod} = (1/12) ML^2 \quad I_{CM}^{disk} = (1/2) ML^2 \quad I_{CM}^{sphere} = (2/5) ML^2 \quad I = I_{cm} + Md^2$$

$$\mathbf{p} = m\mathbf{v} \quad \mathbf{F} = \frac{d\mathbf{p}}{dt} \quad \Delta\mathbf{p} = \mathbf{F}_{av}\Delta t \quad \mathbf{L} = I\boldsymbol{\omega} \quad \boldsymbol{\tau} = \frac{d\mathbf{L}}{dt} \quad \Delta\mathbf{L} = \boldsymbol{\tau}_{av}\Delta t$$

$$\sigma = Y \epsilon \quad \frac{F}{A} = Y \frac{\Delta L}{L} \quad \frac{F}{A} = G \frac{\Delta L}{L} \quad \Delta p = -B \frac{\Delta V}{V} \quad Y = 2G(1 + \nu) = 3B(1 - 2\nu)$$

$$F = -kx \quad \tau = -D\theta \quad x = A \cos(\omega t + \phi) \quad x = (A_0 e^{-t/\tau}) \cos(\omega t + \phi)$$

$$\omega = \frac{2\pi}{T} = 2\pi f t \quad T = 2\pi \sqrt{\frac{m}{k}} \quad T = 2\pi \sqrt{\frac{l}{D}} \quad T = 2\pi \sqrt{\frac{I}{MgL}} \quad T = 2\pi \sqrt{\frac{L}{g}}$$

$$D(x,t) = A \sin(kx - \omega t + \phi) \quad D(x,t) = A \sin(kx) \cos(\omega t) \quad k = \frac{2\pi}{\lambda} \quad \omega = \frac{2\pi}{T} = 2\pi f t \quad v = \lambda f = \frac{\omega}{k} \quad v = \sqrt{\frac{T}{\mu}} \quad v = \sqrt{\frac{Y}{\rho}}$$

$$W = \mathbf{F} \cdot \Delta \mathbf{x} \quad W = \int_{x_1}^{x_2} F(x) dx \quad \Delta K + \Delta U = W_{nc} \quad \Delta K + \Delta U = 0 \quad K = \frac{1}{2} mv^2 \quad K = \frac{1}{2} I\omega^2$$

$$\Delta U_g = mgh \quad \Delta U_g = -\frac{GMm}{R} \quad \Delta U_s = \frac{1}{2} kx^2$$

$$F = \sqrt{F_x^2 + F_y^2 + F_z^2} \quad F_x = F \cos \alpha, \quad F_y = F \cos \beta, \quad F_z = F \cos \gamma \quad \cos^2 \alpha + \cos^2 \beta + \cos^2 \gamma = 1$$

$$g = 9.81 \text{ m/s}^2 \quad G = 6.67 \times 10^{-11} \text{ N m}^2/\text{kg}^2 \quad \bar{x} = \frac{\sum_i x_i}{N} \quad \sigma = \sqrt{\frac{\sum_i (x_i - \bar{x})^2}{N-1}} \quad \sigma_{\bar{x}} = \frac{\sigma}{\sqrt{N}} = \sqrt{\frac{\sum_i (x_i - \bar{x})^2}{N(N-1)}}$$

$$\frac{d(x^n)}{dx} = nx^{n-1} \quad \frac{d(\cos x)}{dx} = -\sin x \quad \frac{d(\sin x)}{dx} = \cos x$$